

INVESTICE DO ROZVOJE VZDĚLÁVÁNÍ

Robustné PID-regulátory s obmedzeniami

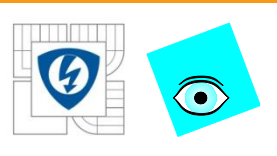
Robust Constrained PID Control

Introduction

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Ing. Peter Ťapák, Ph.D.

Tato prezentace je spolufinancována Evropským sociálním fondem a státním rozpočtem České republiky.



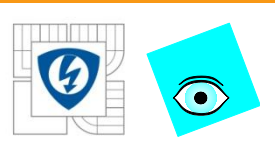
Overview

- Introduction
- Motivation & Terminology
- Table of fundamental controllers
- Conclusions

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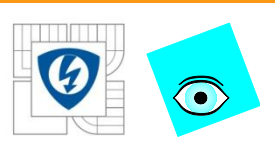
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Control Design

- Early/Heuristic/PID Control/Anti Windup control
- Minimum Time/Fuel control
- Linear Pole Assignment/Optimal Control (LQ, LQG,...), Predictive Control
- Adaptive Control, Nonlinear Control (differential/algebraic/GS,...)
- Robust Control (IMC,...), Fuzzy & Neural Control
- Constrained Control (ellipsoid sets, LMI, ...), etc.
- **Just a Limited Integration in Literature**
- **Needs for a unifying approach**

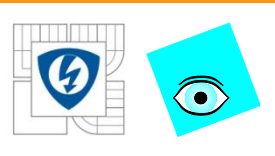


PID Controller architecture

- O'Dwyer – 7 PI & 47 PID controller structures
- Which one is the “standard” one?
- What is the PID controller?
- Here, controller derived for the plants

$$S(s) = \frac{b_1 s + b_0}{s^2 + a_1 s + a_0} e^{-T_d s}$$

- Dead-time compensators for systems with long dead-time are included

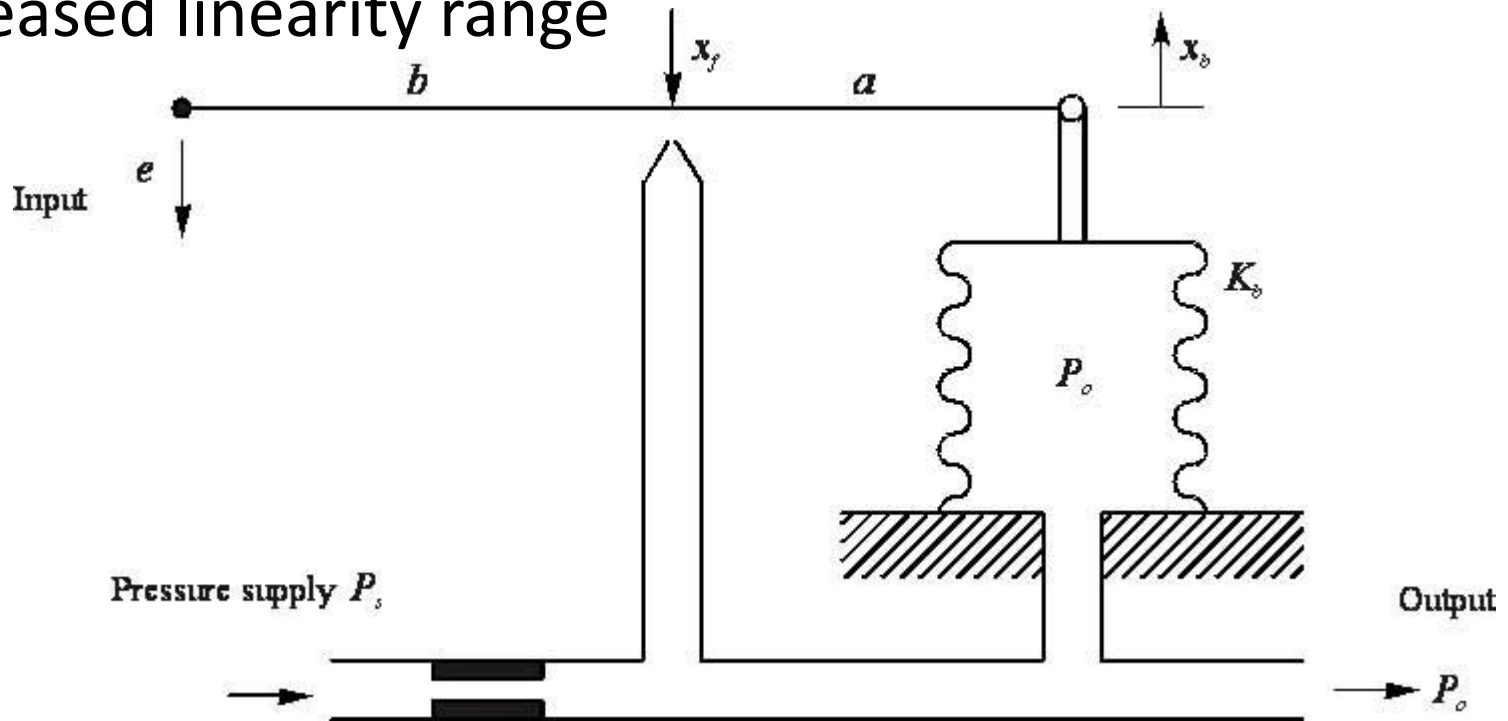


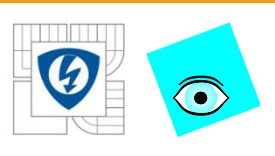
Pneumatic P-Controller (1918)

Milk pasteurization

Negative **feedback** from the controller output

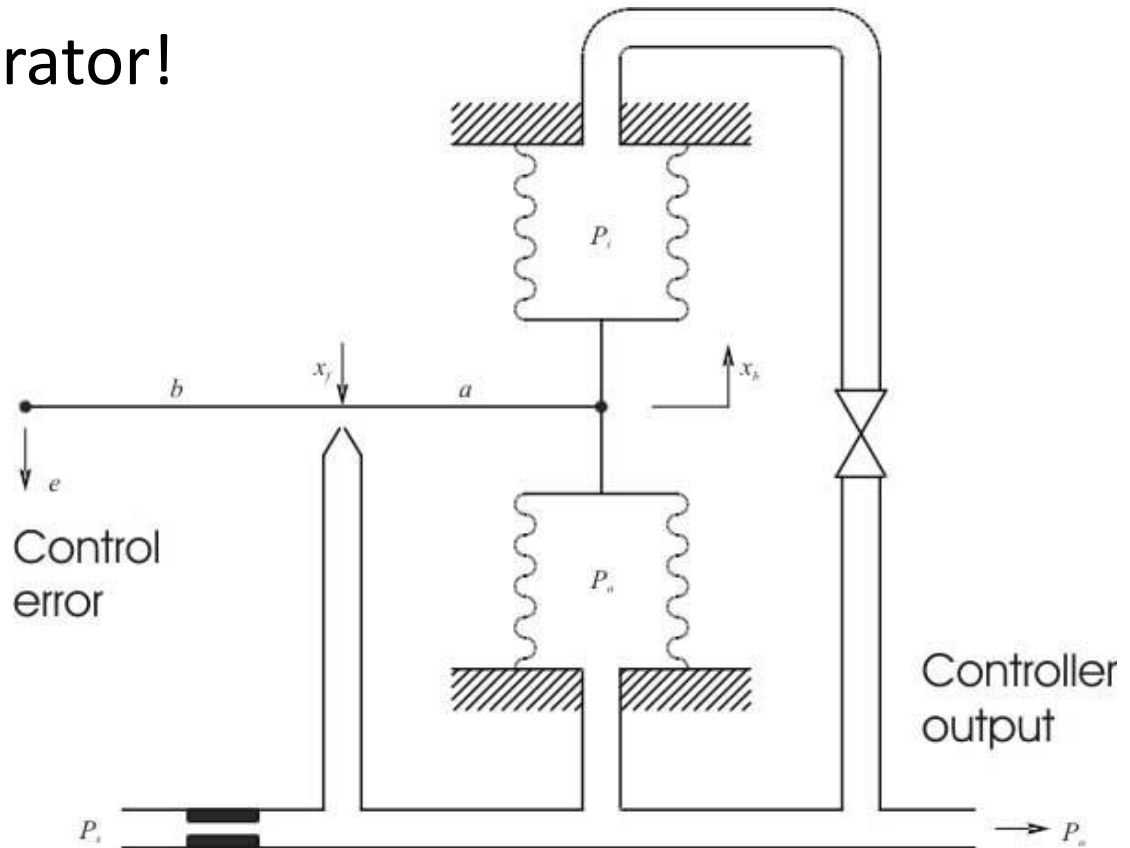
Increased linearity range





Pneumatic PI Controller (1918)

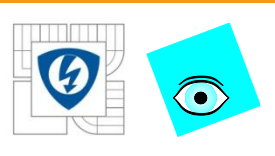
- Integral (I) action called “AUTOMATIC RESET”
- No explicit integrator!
- No windup!



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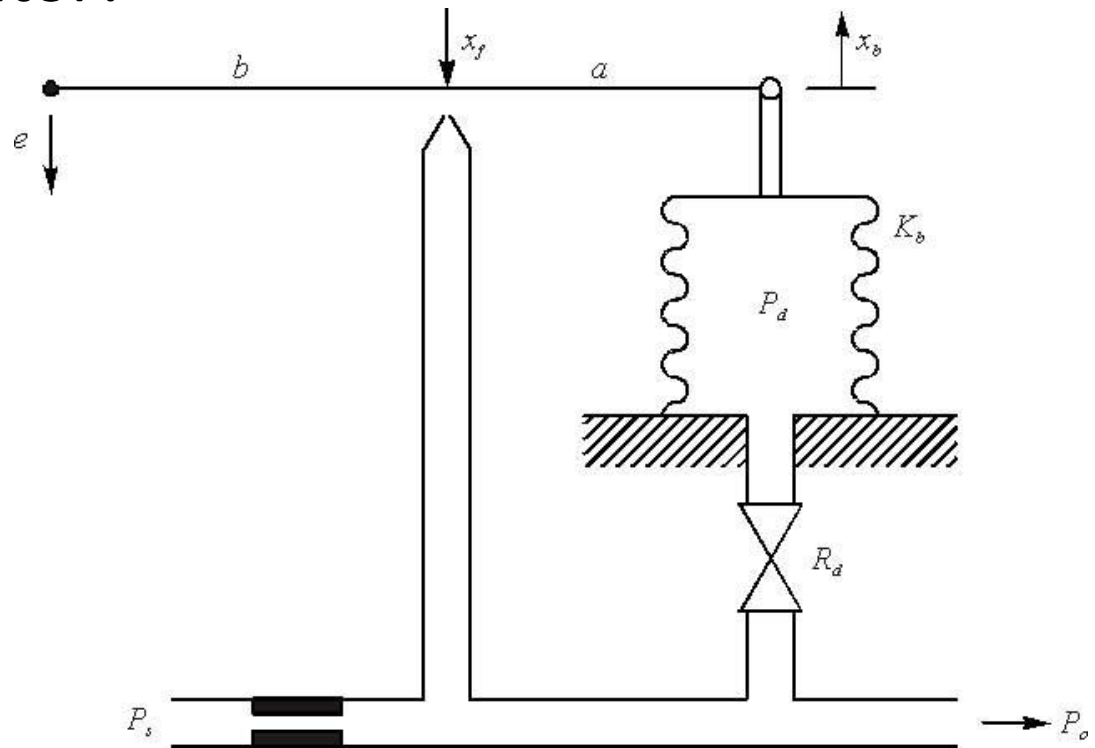
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Pneumatic PD Controller (1935)

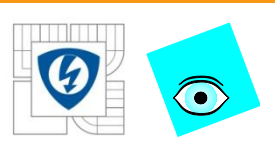
- D action called 'PRE-ACT'
- No explicit derivator!





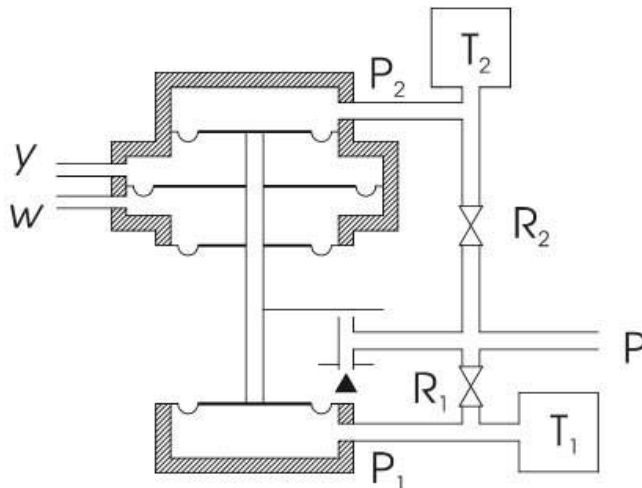
- PRE-ACT
and derivator!
-
- The diagram shows a hydraulic system with a feedback loop and a pre-actuator. A horizontal line represents a feedback path with a gain b and a delay a . The error signal e is the difference between the reference x_r and the output x_b . The output x_b is the displacement of the upper piston. The system consists of two cylinders: an upper cylinder with pressure P_1 and a lower cylinder with pressure P_2 . The lower cylinder is connected to a reservoir at pressure P_r through a valve R_2 . The upper cylinder is connected to the lower cylinder through a valve R_1 . The output P is the pressure in the lower cylinder.

8

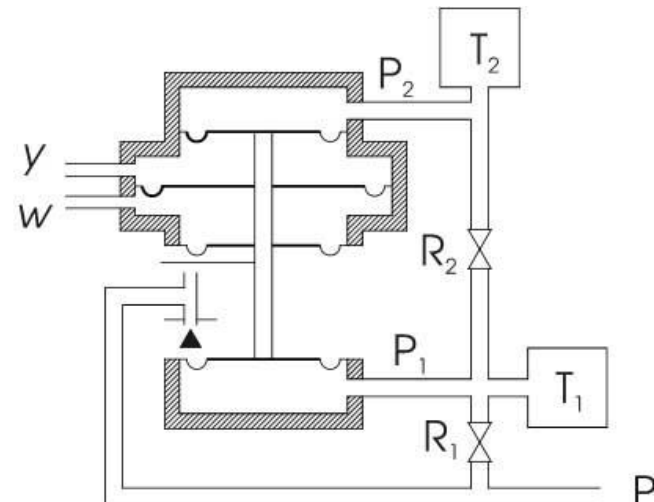


Pneumatic PID Controller (1938)

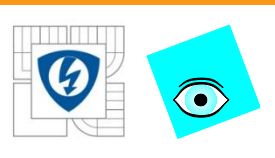
- AUTOMATIC RESET & PRE-ACT
- Diaphragm type controllers
- Different realizations:
 - Different range of adjustable parameters
 - Different properties and controller tuning



Parallel feedback



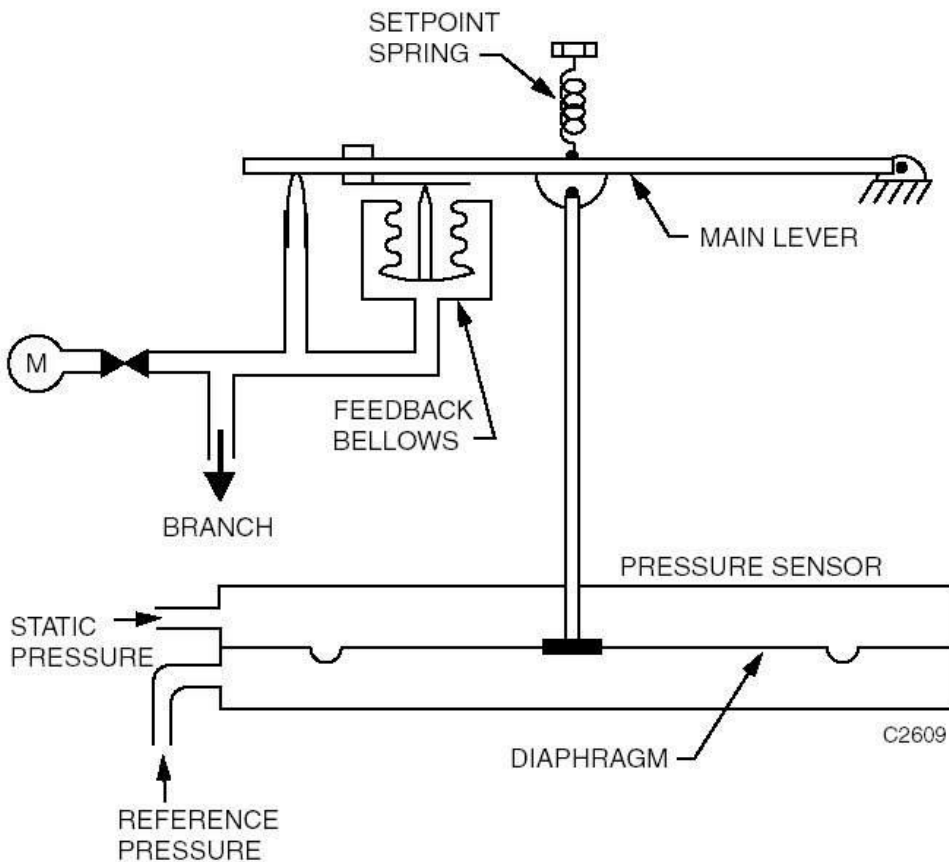
Serial feedback



Pneumatic PI Controller *HONEYWELL*

<http://customer.honeywell.com/Techlit/Pdf/77-0000s/77-E1100.pdf>

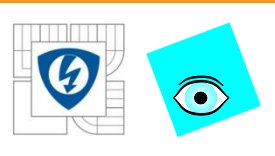
Solution combining
diaphragm and bellows



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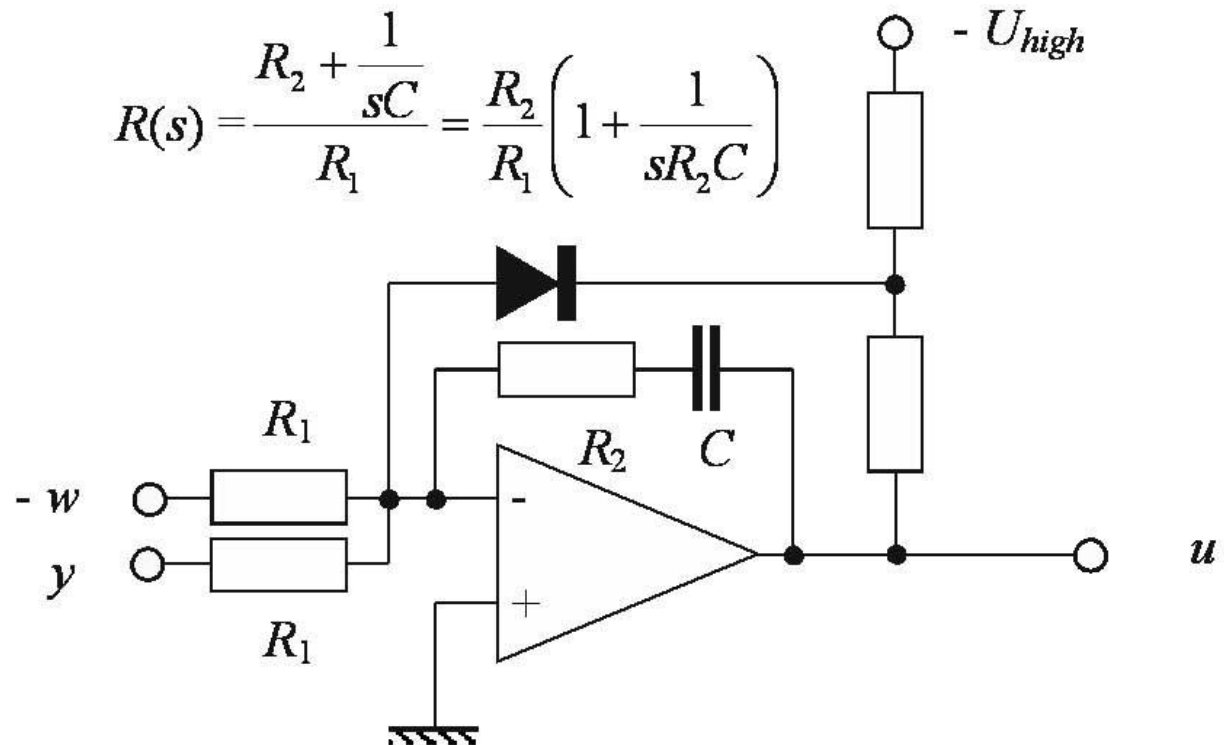
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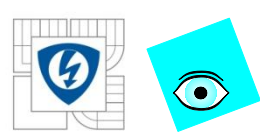




Electronic PI Controller

- No explicit integrator!
- Modular design based on **feedback** from the controller output
- No windup!





Explicit integrator

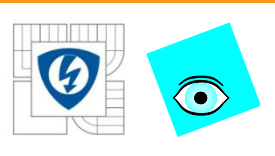
- Involved in hydraulic & electromechanical actuators
- Mechanical constraints for integration
- No windup



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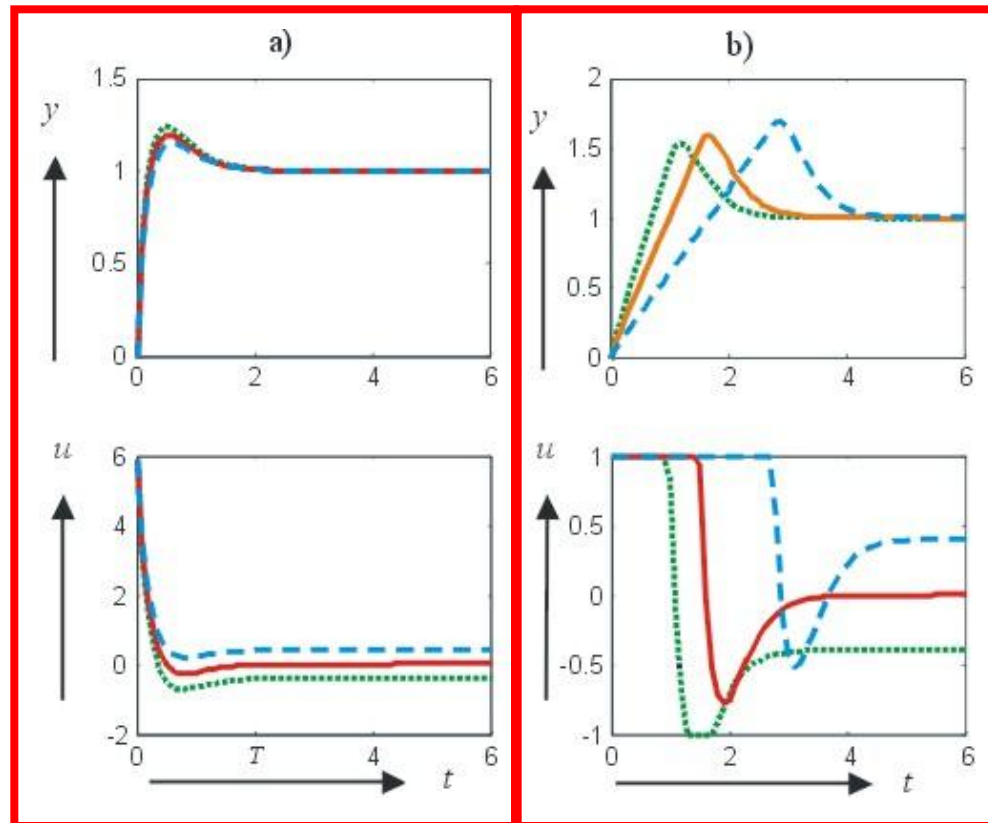


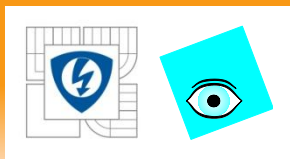
μ -Processor Based PID control

- Use of explicit digital integrators and derivators
- No natural control constraints
- **Windup Problem!**

a) Linear Control
Typical overshooting

b) Constrained Control
Note increased output
overshooting!





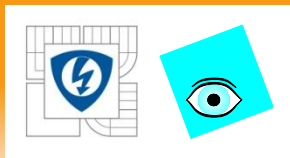
1990 – today:

Revival of Constrained Control

Henrion, Tarbouriech and Kučera (2001)

Control of linear systems subject to input constraints: a polynomial approach. Automatica 37, 597-604.

- implicit (a posteriori) methods - anti-windup strategies
- explicit (a priori) methods divided into
 - saturation avoidance and
 - saturation allowance methods

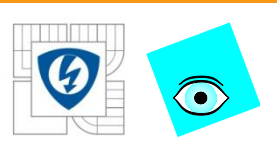


1990 – today:

Revival of Constrained Control

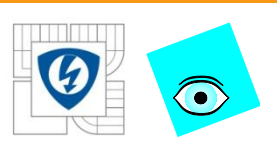
Goodwin (2001), Rojas & Goodwin (2002)

- **cautious** approach = reducing demand on control performance until constraints are avoided (worst case approach $\leftrightarrow$ explicit saturation avoidance)
- **evolutionary** linear design + compensation of constraints $\leftrightarrow$ implicit one - anti-windup strategies
- **tactile** approach $\leftrightarrow$ the explicit allowance one - constraints included in control design from beginning - potentially improved performance, often at the expense of **increased complexity**.



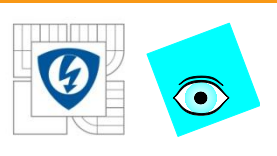
Anti-Windup (Anti-Reset-Windup)

- The most frequent approach
- No generally accepted windup definition!!!
- No homogenous dynamics for large and small steps
- **More complex structures with increased number of parameters**



Saturation Allowance-Tactile Approach

- Gain scheduled controllers, invariant sets, piecewise linear (affine) control, LMI, MPC
- Ellipsoidal Technique, Polyhedral Sets, Piecewise Quadratic Estimates of Attraction
- Robust Constrained PID Control – attractive alternative for simple problems

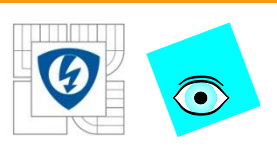


Robust Constrained PID Control

Considering

- properties of classes of solutions,
- limited information about the process,
- process properties changing with time or with the operating point
- constraints put on the process input, state and output and on the rate of their changes,
- variety of requirements of practice,
- development of our knowledge.





Robust Constrained PID Control

Learning by doing supported by:

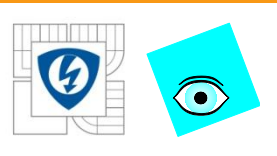
- MS PowerPoint and Java presentations,
- e-books, Flash and Java animations,
- simulation programmes Matlab/Simulink,
- programmes in Maple V,
- assignments to be solved,
- quises for self-testing with feedback,
- manuals for plant models and real-time experiments,
- all integrated in LMS Moodle,
- + printed textbook/workbook



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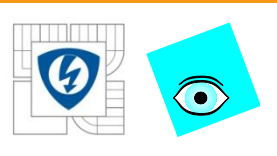




Robust Constrained PID Control

- Fundamental Solutions
- Dynamical Classes of Control
- Table of fundamental controllers (P, PD, I, PI, PID & Predictive P, PD, I, PI and PID controllers)
- Performance requirements
- Robust controller tuning - performance portrait

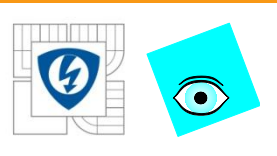




Control Strategies

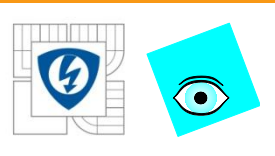
Close relations to:

- Minimum Time Control
- Pole Assignment Control
- State Space Approach (Disturbance Observer)
- IMC (Internal Model Control),
- Dead Time Compensators



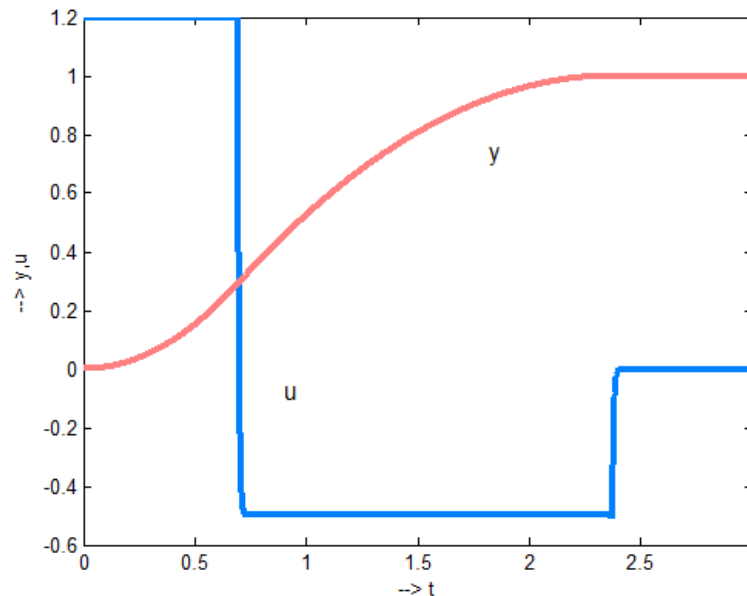
Control Strategies

- All integrated within:
- Robust Constrained Pole Assignment Control
- Basic building block:
- Fundamental controller (solution)

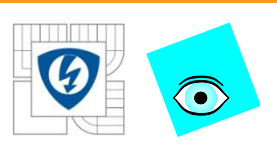


Minimum Time Control

- Bringing system from an initial state to the required one by constrained control $U_1 \leq u \leq U_2$ in the minimum possible time

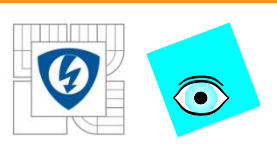


Double
integrator



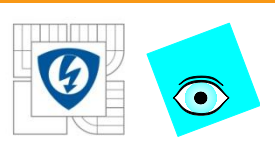
Minimum Time Control

- Less attention usually devoted to **constrained rate** of control changes and **systems with zeros**
- Ad Hoc (**fixed**) **ideal** solution (nominal dynamics, limited possibility to respect parasitic phenomena)
- **High sensitivity** to noise, unmodelled dynamics, parameters fluctuations - relay chattering etc.
- Not solved **steady states**



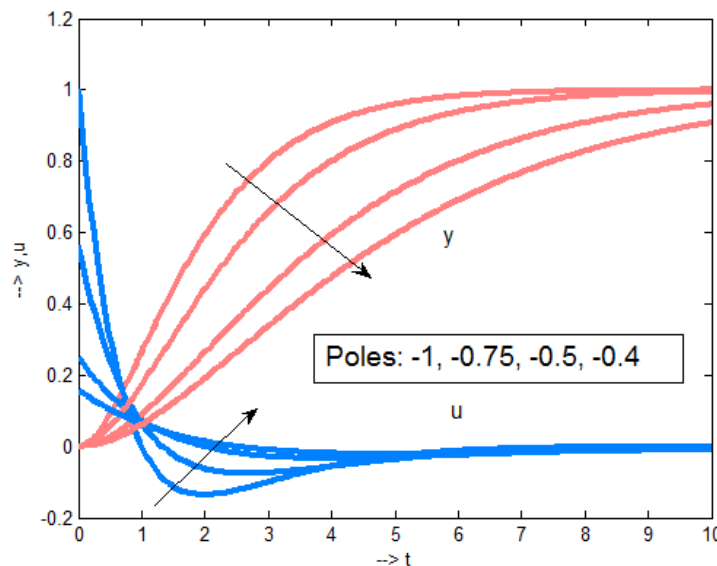
Minimum Time Control

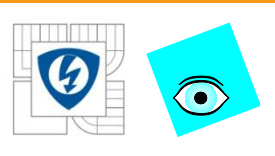
- Usually not directly applicable in process control
- Theoretical value - giving limits for achievable control dynamics with:
 - no noise,
 - no nonmodelled dynamics,
 - no parameter fluctuations
- Focused on ideal control with no uncertainty



Linear Pole Assignment Control

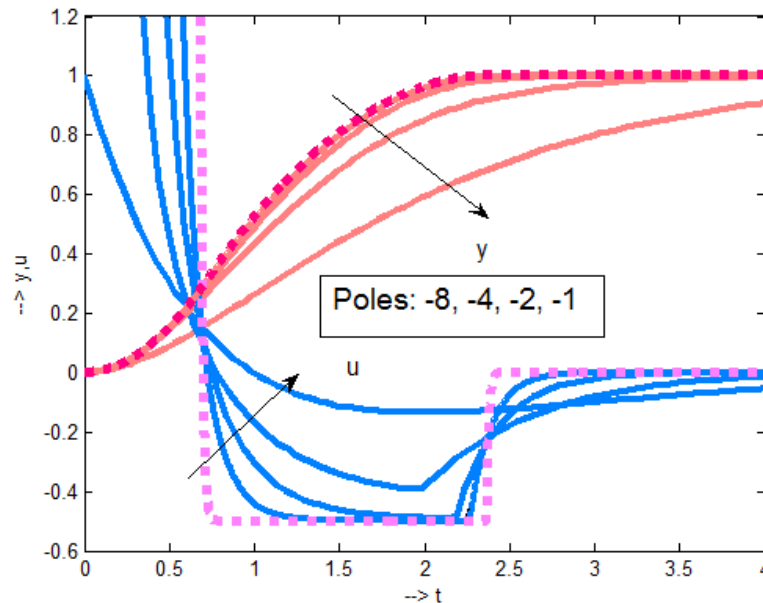
- **Continuum of solutions** expressed in terms of poles & polynomials
- Known effect of poles shifting



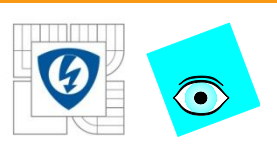


Constrained Pole Assignment Control

- Bringing system from an initial state to the required one by **constrained control** $U_1 \leq u \leq U_2$ & possible rate constraints
- **Continuum of solutions** expressed in terms of poles & polynomials – MTC with additional constraints

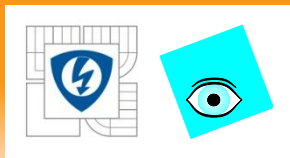


Double
integrator



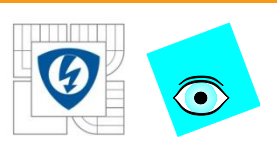
Linear Pole Assignment Control

- Experimental - “trial and error approach” (noise, constraints, nonmodelled dynamics, parameter fluctuations etc.)
- Not able to respond to the basic question “slow or fast ” poles?
- RCPIDC: Offering poles depending on measurement noise, substantial time delays, parameter fluctuations, control signal constraints.



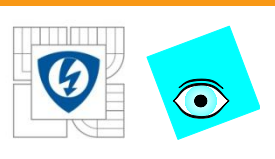
Fundamental solutions in RCPIDC

- Parameterized solutions with dynamics scaled by the closed loop poles
- Dominant dynamics - monotonic step responses at the plant output with input dynamics ranging from
 - relay minimum time control up to
 - linear pole assignment control
- MTC:
 - n-rectangular pulses of control at the plant input
- RCPAC:
 - dynamical classes with nP control signal



Dynamical Classes of Control

- Physics in the control
- Felbaum's theorem about n -intervals of optimal control
- Neglected under linear design $\Rightarrow$ inflation of optimal controller tuning
- Energy accumulation – integrators – nP control
- Time delays – at least one accumulating element with stabilizing feedback
- Time delays – required output may also be achieved by a constant input
- Dead time – infinite number of stable accumulating elements

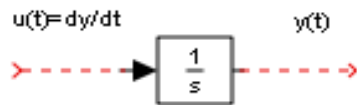


Dynamical Classes of Control

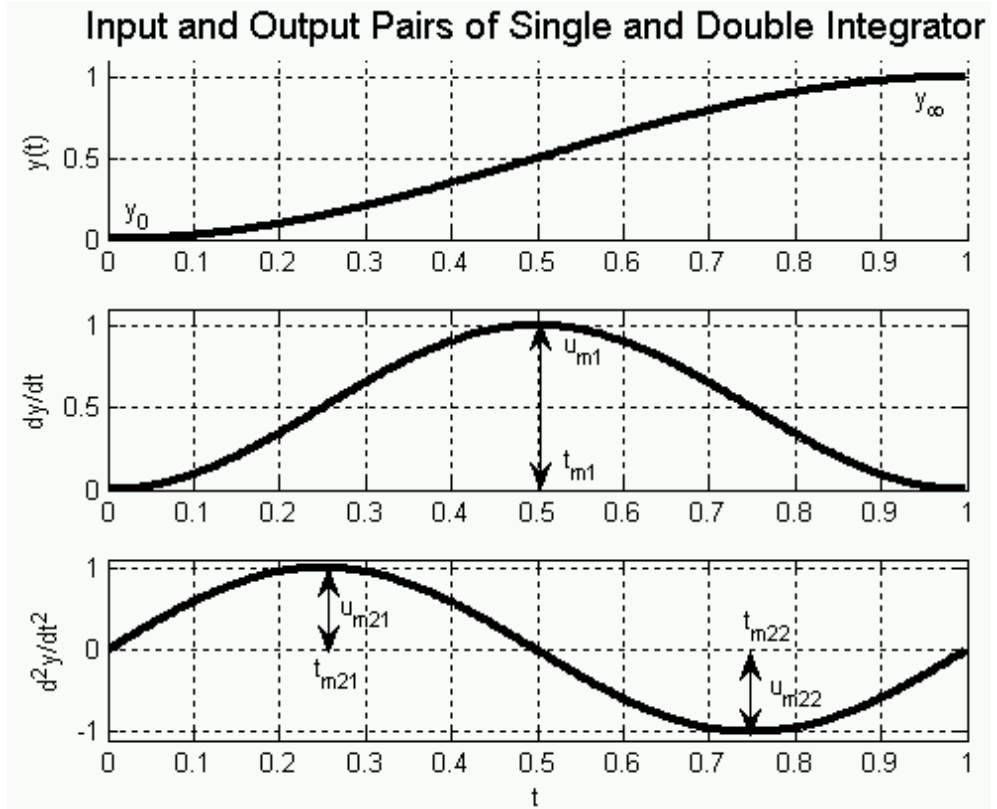
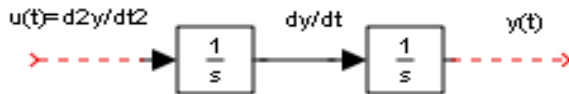
- To achieve **monotonic (0P, MO) output** of the chain of n -integrators we need n pulses of continuous control (**nP control**) at its input

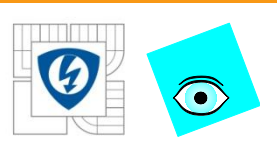
0P (MO) signal $y(t)$

1P signal $u(t)$



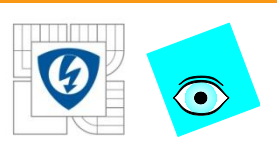
2P signal $u(t)$





Dynamical Classes of Control

- Dynamical class of control DCn – all solutions and control structures with MO output and nP input
- Traditional PID control design – dominated by the DC0 and DC1 (robustness, sensitivity to noise, complexity, windup problems)
- DC2 – approached e.g. by the Model Predictive Control (MPC)
- For stable plants, the Dynamical Class of control may be decreased up to 0
- Index of the DC cannot be decreased below the number of unstable poles



Dynamical Classes of Control

- Table of fundamental controllers
- Similarity with the Mendelejev table
- Similar properties of classes of controllers
- Controllers classified:
 - With respect to the DC of control
 - With respect to the plant approximation
- Each controller may exist in a form allowing discontinuity of control at the origin and in a filtered version with a fully continuous control

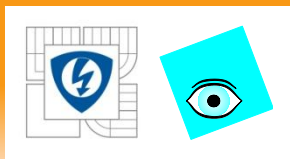


Table of Fundamental Solutions

Dynamical Class 0

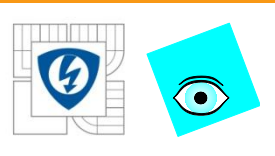
		Dominant dynamics							
Dynamic class	I-action	K	Ke^{-T_d}	$\frac{K_s}{s+a}$	$\frac{K_s e^{-T_d s}}{s+a}$	$\frac{K_{s1}}{s+a_1} + \frac{K_{s2}}{s+a_2}$	$\left[\frac{K_{s1}}{s+a_1} + \frac{K_{s2}}{s+a_2} \right] e^{-T_d s}$	$\frac{K_s}{s^2+a_1 s+a_0}$	$\frac{K_s e^{-T_d s}}{s^2+a_1 s+a_0}$
0	N	FF	FF	FF	FF	FF	FF	FF	FF
	Y	I	PrI	PI	PrPI	PID	PrPID	PID	PrPID
1	N	-	-	P	PrP	P-P	PrP-P	PD	PrPD

Explicit - Saturation Avoidance Approach

	Y	-	-	-	-	-	-	PID	PrPID
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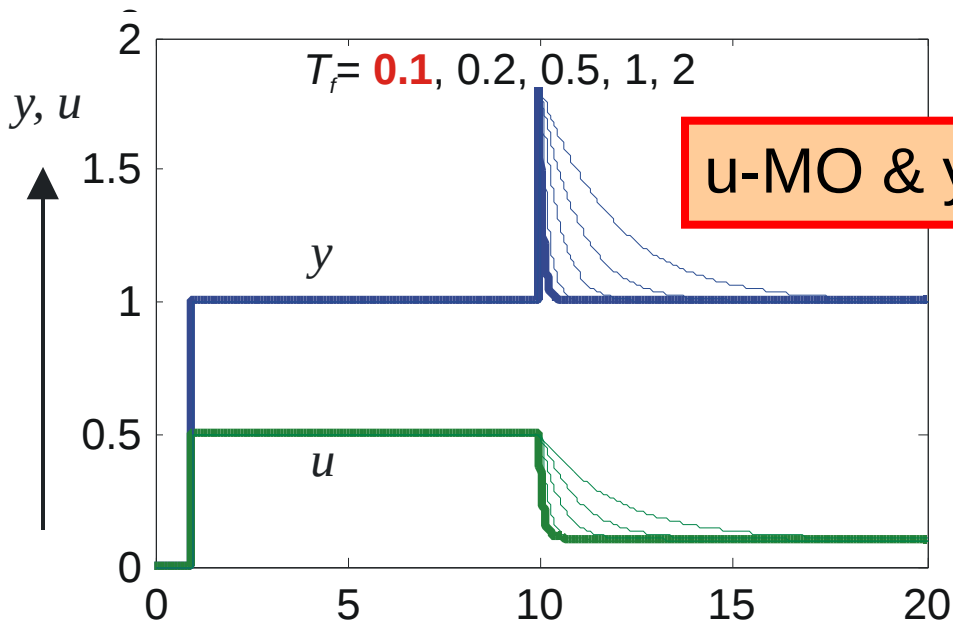
FF = static feedforward control

Pr = predictive (dead time) controller



Fundamental Solutions DC0

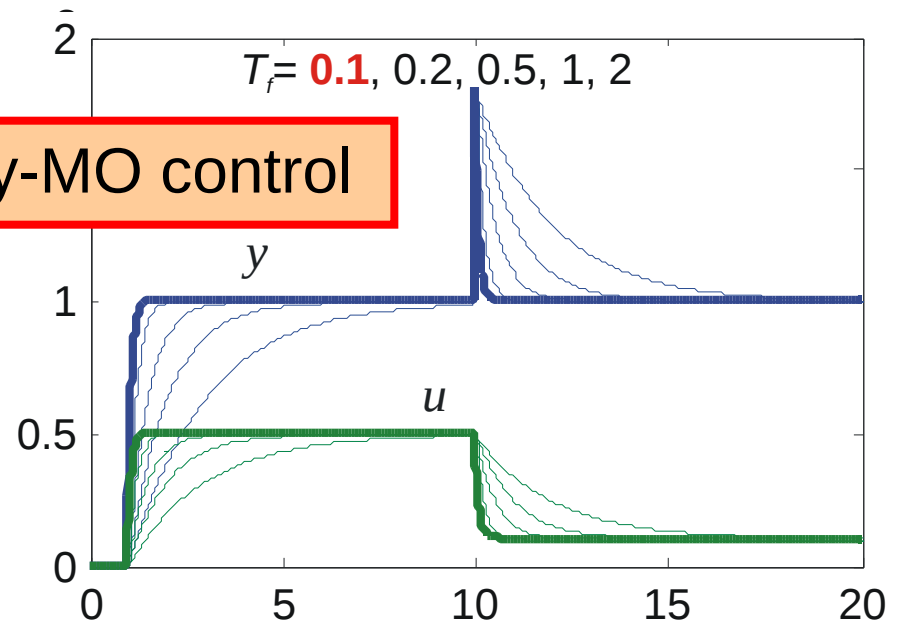
Continuous for $t > 0$



u-MO & y-MO control

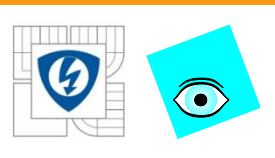
Basic Controllers

Continuous for $\forall t$



Filtred Controllers

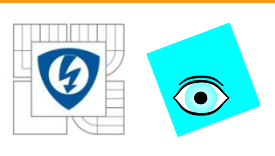
Parametrization by the closed loop time constant ($T_f = -1/\alpha$, α - closed loop pole)



Fundamental Solutions DC1

Dominant dynamics									
Dynamic class	I-action	K	$Ke^{-T_d s}$	$\frac{K_s}{s+a}$	$\frac{K_s e^{-T_d s}}{s+a}$	$\frac{K_{s1}}{s+a_1} + \frac{K_{s2}}{s+a_2}$	$\left[\frac{K_{s1}}{s+a_1} + \frac{K_{s2}}{s+a_2} \right] e^{-T_d s}$	$\frac{K_s}{s^2+a_1 s+a_0}$	$\frac{K_s e^{-T_d s}}{s^2+a_1 s+a_0}$
0	N	FF	FF	FF	FF	FF	FF	FF	FF
	Y	I	PrI	PI	PrPI	PID	PrPID	PID	PrPID
1	N	-	-	P	PrP	P-P	PrP-P	PD	PrPD
	Y	-	-	PI	PrPI	P-PI	PrP-PI	PID	PrPID
2	N	-	-	-	-	-	-	PD	PrPD
	Y	-	-	-	-	-	-	PID	PrPID

Explicit - Saturation Allowance Approach

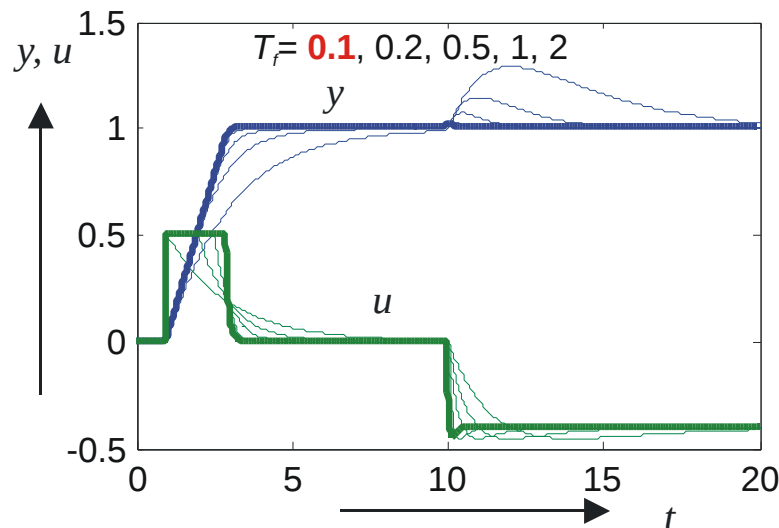


Fundamental Solutions DC1

u-1P & y-MO control

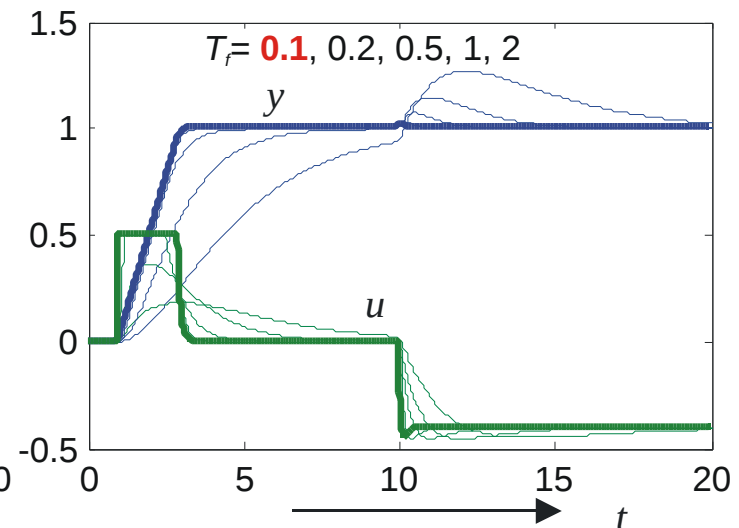
Basic Controllers

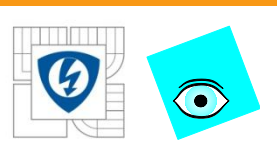
Continuous for $t > 0$



Filtred Controllers

Continuous for $\forall t$

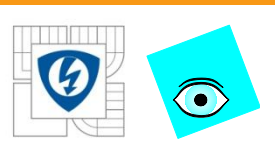




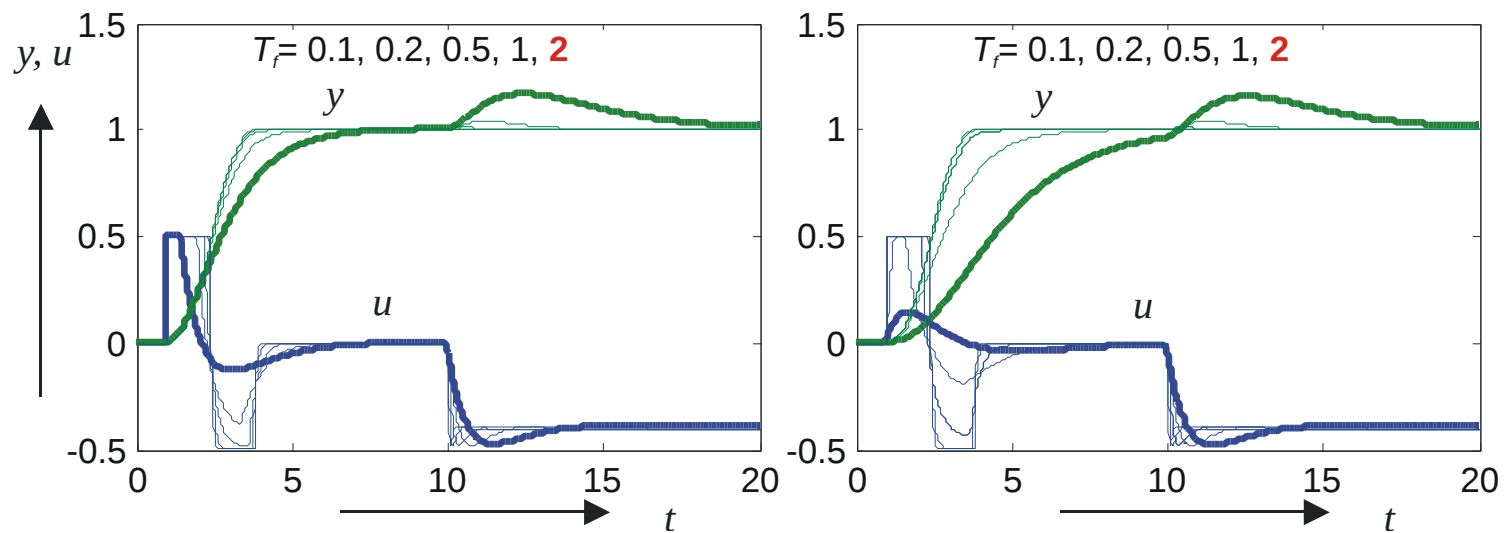
Fundamental Solutions DC2

		Dominant dynamics							
Dynamic class	I-action	K	Ke^{-T_d}	$\frac{K_s}{s+a}$	$\frac{K_s e^{-T_d s}}{s+a}$	$\frac{K_{s1}}{s+a_1} + \frac{K_{s2}}{s+a_2}$	$\left[\frac{K_{s1}}{s+a_1} + \frac{K_{s2}}{s+a_2} \right] e^{-T_d s}$	$\frac{K_s}{s^2+a_1 s+a_0}$	$\frac{K_s e^{-T_d s}}{s^2+a_1 s+a_0}$
0	N	FF	FF	FF	FF	FF	FF	FF	FF
	Y	I	PrI	PI	PrPI	PID	PrPID	PID	PrPID
1	N	-	-	P	PrP	P-P	PrP-P	PD	PrPD
	Y	-	-	PI	PrPI	P-PI	PrP-PI	PID	PrPID
2	N	-	-	-	-	-	-	PD	PrPD
	Y	-	-	-	-	-	-	PID	PrPID

Explicit - Saturation Allowance Approach



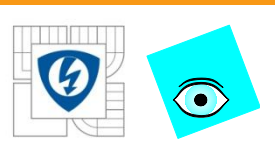
Fundamental Solutions DC2



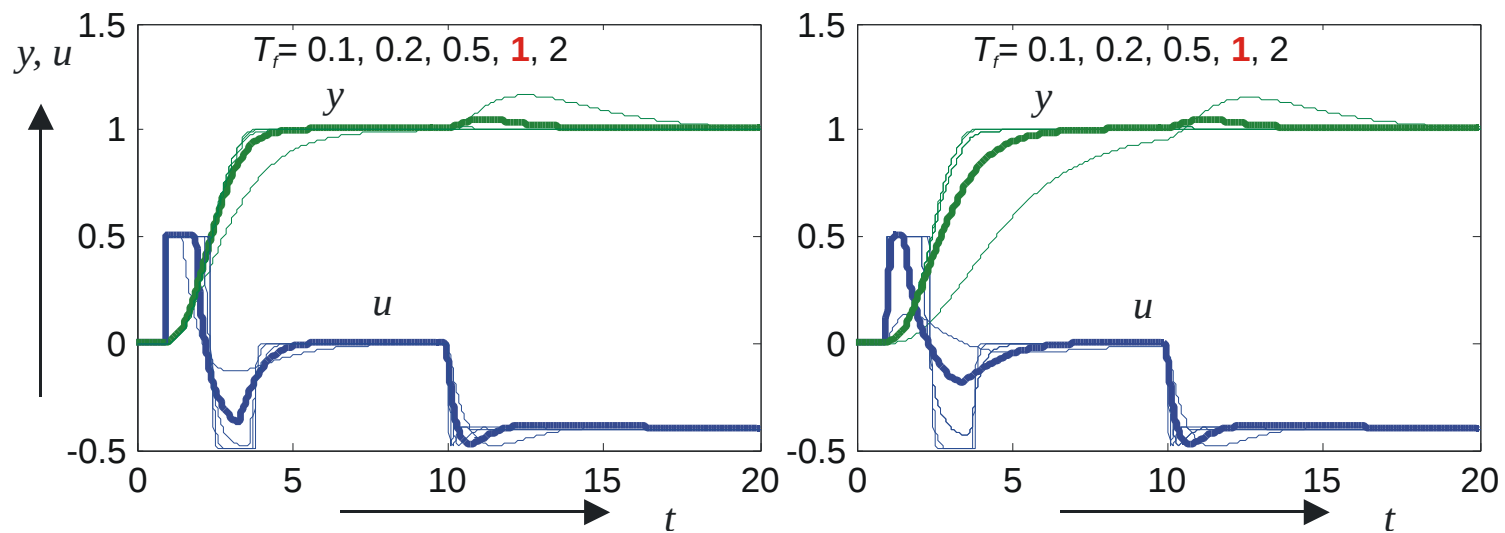
11.3.2011

INVESTICE DO ROZVOJE VZDĚLÁVÁNÍ





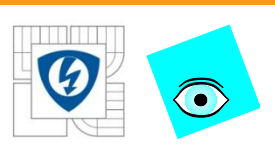
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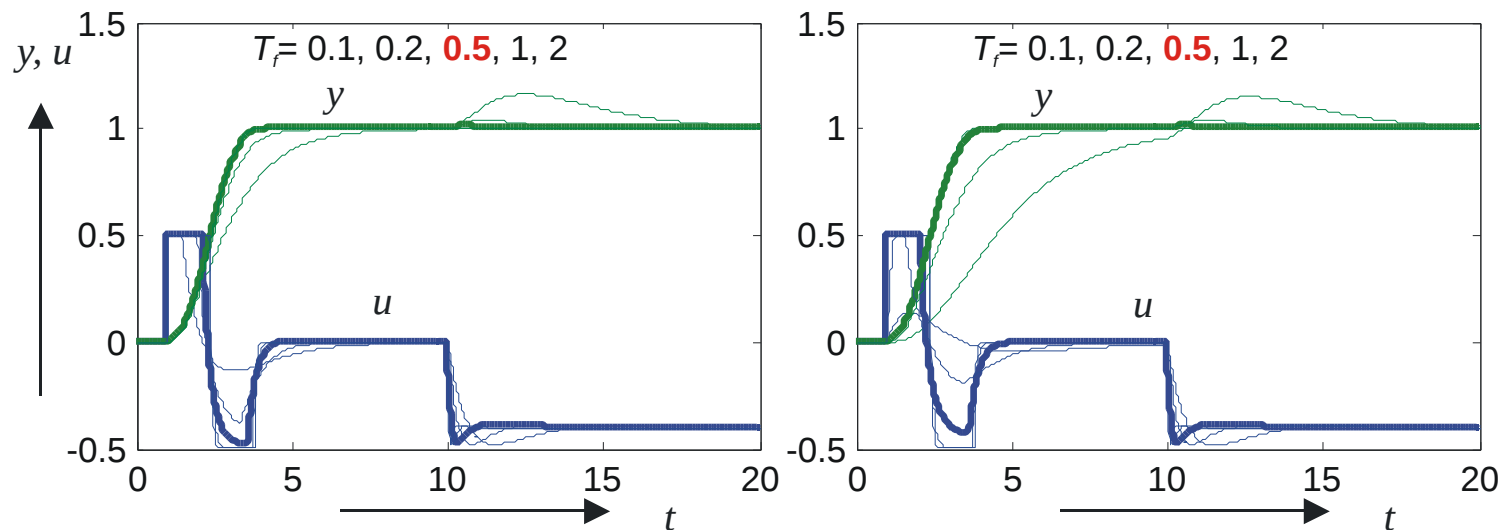
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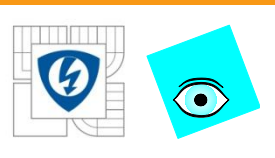
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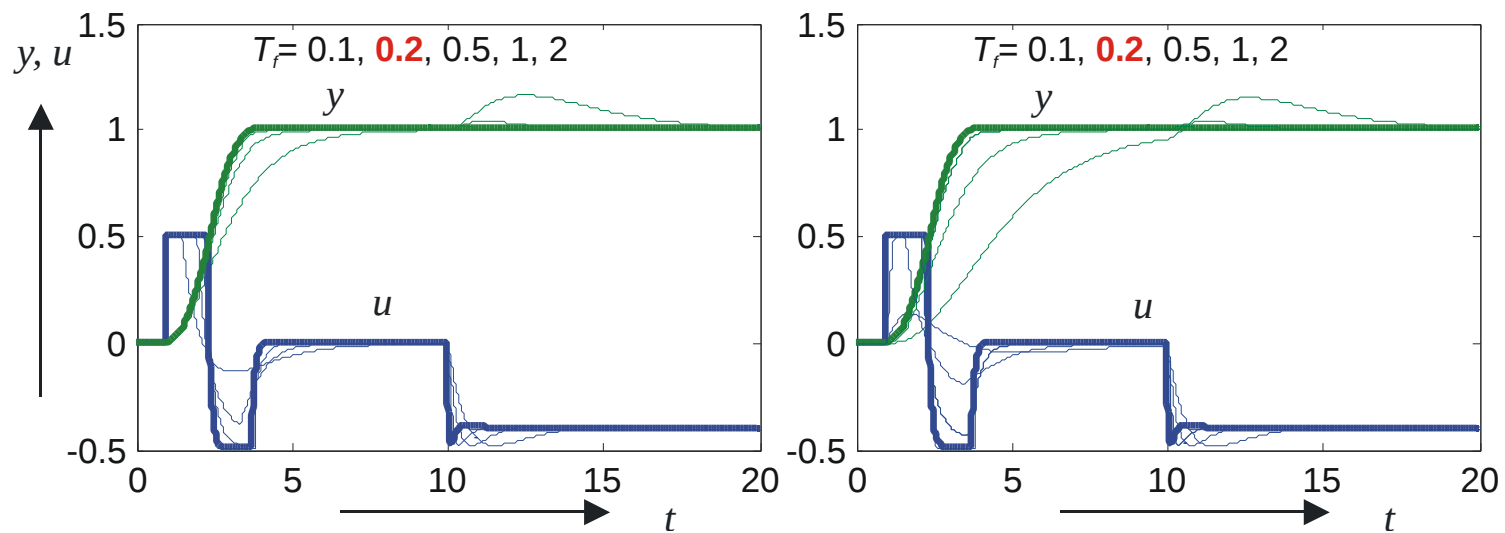
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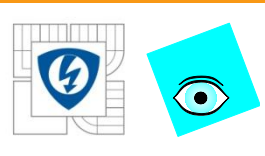
Fundamental Solutions DC2



11.3.2011

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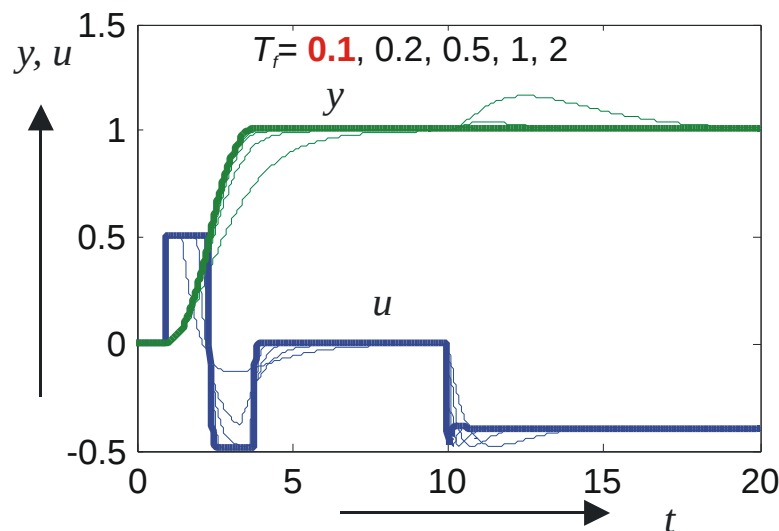




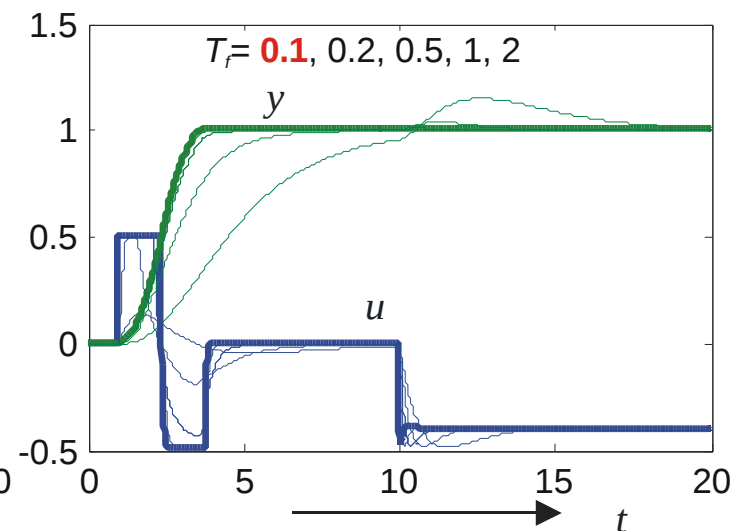
Fundamental Solutions DC2

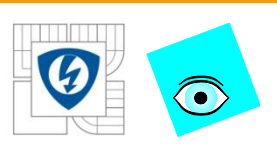
u-2P & y-MO control

Basic Controllers



Filtred Controllers

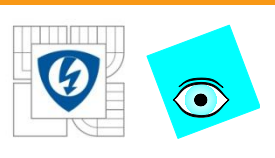




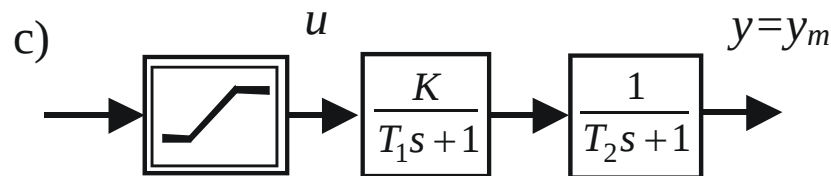
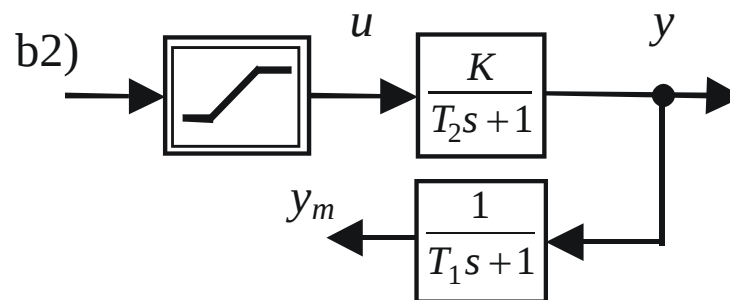
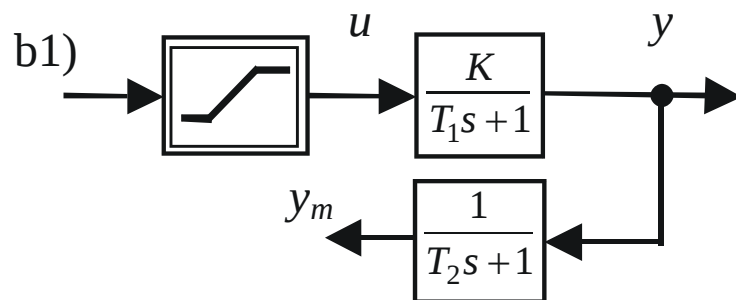
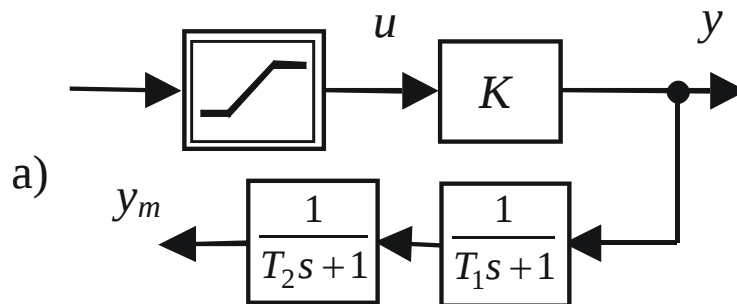
Fundamental Solutions

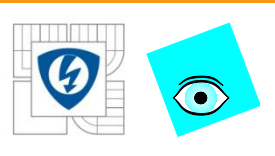
- **2** Dynamical Classes of PI Controller
- **3** Dynamical Classes of PID Controller
- **4** Basic Situations in PID Tuning!

Loop Dynamics Distribution



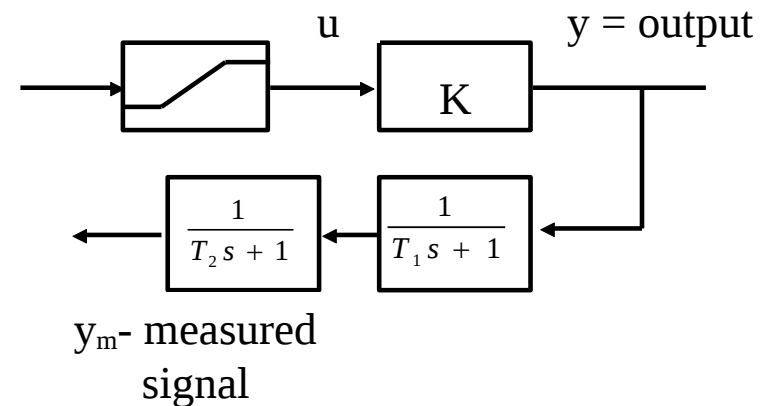
Loop Dynamics Distribution

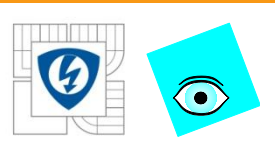




DC0: Dominant Feedback Dynamics

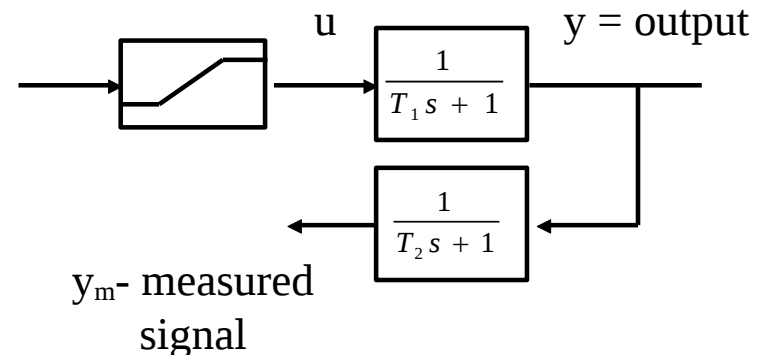
- y-MO and u-MO control transients
- Optimal control corresponding to a reference step has a step character
- Linear design can be used
- Minimal Noise Sensitivity

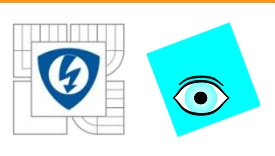




DC1:1st Order Relative Degree

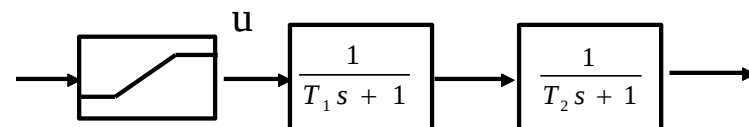
- y -MO and u -1P control transients
- Modified Anti Windup approach can be used
- Several structures denoted as PI_1 , or PID_1
- Higher noise sensitivity



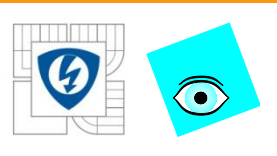


2nd Order Relative Degree

- y -MO and u -2P control transients
- Two intervals of control
- Many Anti Windup approaches fail in the case of unstable (marginally stable) plant dynamics
- Several structures denoted as PD_2 , or PID_2
- Highest noise sensitivity



$y = y_m$
output=
measured
signal



Conclusions

- To deal effectively with the control constraints, each Dynamical Class of control requires special control structures, despite to the fact that the loop may be approximated by the same model
- Conclusion - for the 1st order loop we need two dynamical classes of the PI control and for the 2nd order dominant loop dynamics there exist three DCs of the PID control
- DC index n is determined by the output relative degree, but for stable plant poles it may be decreased (up to the number of unstable poles)